

## SEALING DEVICE FOR A WHEEL HUB GROUP

### DESCRIPTION

The present invention relates to a sealing device for a wheel hub group.

5       The present invention may be advantageously, but not exclusively, applied in the field of wheel hub groups which are connected to the differential of a vehicle, and provided with rolling bearings, and in which the sealing device is mounted in such a way as  
10 to protect the rolling bearing on an inner side of the bearing, or rather on a side of the bearing which is turned towards the differential.

The description which follows will refer, in the interest of providing an example, to this  
15 specific application, without however losing any of its general nature.

In the application which has just been described above, the wheel hub group and the differential are connected to each other by means of  
20 an axle shaft, which is arranged inside a sealing housing which extends from the differential as far as the wheel hub group, and which is substantially immersed in a lubricating fluid which is contained inside the sealing housing for the lubrication of the  
25 differential and the axle shaft themselves.

Furthermore, the sealing device comprises, in its more generic form, a shield which is force fit onto an outer race of the bearing, a second shield which is force fit onto an inner race of the bearing and  
5 which faces the first shield, and a dynamic sealing element which is interposed between the first and second shields.

The recent technological advances in the field of wheel hub groups have lead to the implementation  
10 in such groups of devices, such as encoders, for indicating the kinematic functioning parameters of the wheel hub groups themselves, such devices normally being integrated into the above described sealing devices in such a way that they are  
15 rendered integral to one of the two shields, generally the more outer of the two shields in relation to the bearing.

In the above-described application, however, it has been revealed that the metallic contaminating  
20 agents which are inevitably to be found in the lubricating fluid can cause some disadvantages which, in some cases, can have a detrimental effect on the quality of the signal which is emitted by the encoder.

25 The aim of the present invention is to produce

a sealing device for a wheel hub group, which will permit the use of an encoder in an application such as the one described above, but which will not present the disadvantages which have been indicated  
5 above.

According to the present invention, a sealing device will be produced for a wheel hub group connected to a differential device, and provided with a rolling bearing, the sealing device being mounted  
10 in such a way as to protect the bearing from a lubricating fluid for the lubrication of the differential, and comprising a first shield which is integral with an outer race of the bearing, a second shield which is integral with an inner race of the  
15 bearing and which faces the first shield, and a dynamic sealing element which is interposed between the first and second shields; the sealing device being characterised by the fact that the second shield is arranged internally to the first shield in  
20 relation to the bearing, and comprises a support portion which is made of metallic material and which is force fit onto the inner race and an external portion which is provided with a cylindrical encoder which is integral with the support portion; the first  
25 shield comprising a first cylindrical portion which

is made of metallic material and which is force fit  
onto the outer race in a position which is at least  
coaxial to the encoder, and which is provided with at  
least one slit which is suitable for being engaged by  
5 a sensor for reading a signal which is generated by  
the encoder itself.

The invention will now be described with  
reference to the attached drawings, which illustrate  
a non-limiting form of embodiment of the present  
10 invention, and in which:

- FIGURE 1 is a section view, with some parts  
schematised for reasons of clarity, of a preferred  
form of embodiment of the sealing device for a wheel  
hub group,

15 - FIGURE 2 illustrates, in section and on an  
enlarged scale, a detail of the device shown in  
FIGURE 1;

- FIGURE 3 illustrates, in perspective view and  
on an enlarged scale, a detail of the device shown in  
20 FIGURE 1; and

- FIGURE 4 is a schematised perspective view,  
on a reduced scale and with some parts removed for  
reasons of clarity, of a detail shown in FIGURE 3.

With reference to FIGURE 1, the number one  
25 indicates, in its entirety, a sealing device for a

wheel hub group 2.

The group 2 is suitable for being connected to a vehicle differential 3 by means of an axle shaft 4, which is arranged inside a sealing box/house/case 5 which extends from the differential 3 as far as the group 2, and is substantially immersed in a lubricating fluid which is contained inside the housing 5 for the lubrication of the differential 3 and the axle shaft 4 themselves.

10 The group 2 presents a longitudinal axis A, and comprises a tubular body 6 which is internally crossed by a cylindrical passing housing 7, which is coaxial to the axis A, and is engaged in an axially sliding, but angularly integral, manner by a grooved 15 terminal portion of the axle shaft 4. The group 2 also comprises a sealing plug 8 which is arranged in such a way as to close the housing 7, and a rolling bearing 9, which is mounted on the tubular body 6, and which comprises in turn a fixed outer race 10 and 20 a rotatable inner race 11 which is axially blocked onto the tubular body 6 by means of a rolled blocking border 12 of the tubular body 6 itself.

The race 10 is axially delimited, on the part turned towards the differential 3, or rather the 25 inner part of the group 2, by an outer annular

surface 13 which is transverse to the axis A, and is radially delimited towards the outside, at least on the part turned towards the differential 3, by an outer cylindrical surface 14 which is transverse and  
5 contiguous to the surface 13.

The race 11 axially projects in relation to the race 10, and is axially delimited by an outer annular surface 15 which is transverse to the axis A and arranged against the border 12, and is radially  
10 delimited towards the outside, at least on the part turned towards the differential 3, by an outer cylindrical surface 16 which is transverse and contiguous to the surface 15. The surface 15 is axially staggered in relation to the surface 13,  
15 while both the surface 14 and the surface 16 are coaxial to the axis A, and are engaged by the device 1 in order to block the device 1 itself to the bearing 9, in such a way as to arrange the device 1 so that it protects the bearing 9 from the  
20 lubricating fluid for the lubrication of the differential 3.

As is better illustrated in FIGURES 2 and 3, the device 1 comprises a shield 22 which is integral with the race 10, a shield 23 which is integral with  
25 the race 11 and which faces the shield 22, and a

dynamic sealing element 24 which is interposed between the two shields 22 and 23.

The shield 23 is arranged internally to the shield 22 in relation to the bearing 9, and it  
5 comprises a support portion 25 which is made from metallic material and is force fit onto the surface 16 of the race 11, and an outer portion 26, which is integral with the portion 25, and which is provided with a cylindrical encoder 27, a cylindrical wall 28  
10 which is integral with the encoder 27 and which is radially arranged towards the inside in relation to the encoder 27 itself, and a shaped connection flange 29 between the wall 28 and the portion 25 themselves.

In particular, the flange 29 is transverse to  
15 the portion 25 and presents, on the side of the encoder 27, a substantially tapering wall 30.

The encoder 27 is made of magnetic material, for example elastomeric rubber or plastic, and is radially arranged towards the outside of the walls 28  
20 and 30.

Furthermore, the portion 25 comprises a cylindrical wall 31 which is force fit onto the surface 15, and an external border 32 which is axially arranged towards the outside of the bearing  
25 9, and which has a diameter which is less than a

diameter of the wall 31 in order to define both an axial striker on the race 11 and a static seal on the border 12.

The shield 22 comprises an outer cylindrical wall 33 which is arranged in at least a coaxial position to the encoder 27, and an inner cylindrical wall 34 which is integral with the wall 33, and which is radially arranged opposite the encoder 27 itself in relation to the wall 33 itself. The wall 33 and the wall 34 are both made from metallic material, and they are rendered integral with each other by means of an annular wall 35, which is arranged transverse to the axis A, and which defines with the walls 33 and 34 themselves a toroidal chamber 36 substantially inside which is arranged the encoder 27.

Furthermore, the wall 33 comprises two cylindrical portions 33a and 33b which have different diameters in relation to each other, and an annular connecting portion 33c which connects the two cylindrical portions 33a and 33b, and of which the cylindrical portion 33a is force fit onto the surface 13, and defines with the annular portion 33c a rounded edge 37 which is arranged so as to abut the surfaces 13 and 14. The cylindrical portion 33b, instead, entirely faces the encoder 27, and is



provided with a slit 38 which is suitable for being engaged by a sensor 39 for reading a signal which is generated by the encoder 27 itself. In order to facilitate the mounting of the device 1 onto the bearing 9, the cylindrical portion 33b may however be provided with further slits 38 which are uniformly distributed around the axis A.

The wall 34 presents, on the part opposite the wall 35, a support border 40, which is radially folded towards the axis A, and which defines a support for the element 24.

Finally, the shield 22 comprises a lining 41 made of rubber material, which is arranged outside the shield 22 itself, and which completely lines the annular portion 33c, the cylindrical portion 33b, the annular wall 35, the wall 34, and the border 40, from which it departs in order to define the sealing element 24, which in a case of this kind is defined by two lips 42 and 43, of which the lip 42 axially extends towards the inside of the bearing 9 and is arranged in sliding contact on the wall 31, while the lip 43 axially extends towards the outside of the bearing 9, and is also arranged in sliding contact on the wall 31, but is also provided with an annular spring 44 in order to increase the sealing effect on

the wall 31 itself.

The lining 41 completely occludes the slit 38 by being inserted inside it in order to define a depression 45 which is of a substantially quadrangular shape, which is limited at the bottom by a thin baffle 46 which is made of the same material as the lining, and which is suitable for housing the sensor 39 in its interior. In particular, the baffle 46 directly faces the encoder 27 and the sensor 39 is positioned inside the depression 45 in such a way that one of its own reading surfaces 39a is arranged in direct contact with the baffle 46. In this way, not only is the surface 39a arranged in a position which is very near the encoder 27 with the advantage of minimising the air gap between the sensor 39 and the encoder 27 themselves, but, above all, the surface 39a is also arranged in contact with the baffle 46 by which it is protected from the metallic contaminating agents which are inevitably present in the lubricating fluid.

Finally, the lining 41 comprises a static sealing element 47, which is arranged around the whole edge 37 in order to produce a static seal with the housing 5 in such a way as to also avoid any leaks of lubricating fluid from the housing 5 itself,

and which is defined by a rounded projection which has an outer diameter which is greater than the diameter of the cylindrical portion 33a.

From the above description, it is evident that  
5 the particular conformation of the shields 22 and 23 permit the complete isolation of the encoder 27 from the outside, preventing any contamination whatsoever of the encoder 27 by the metallic contaminating agents which are present in the lubricating fluid,  
10 and equally permitting the protection of at least the reading surface 38a of the sensor 38 itself.

It is intended that the present invention should not be limited to the forms of embodiment  
15 herein described and illustrated, which are to be considered as examples of forms of embodiment of the sealing device for a wheel hub group, which might instead be subject to further modifications relating to the shape and disposition of its parts and details  
20 pertaining to its construction and assembly.